

EXAMPLE 2 "WYE Winding -- Sine / Sine Evaluation"

Conversion factors:

$$\text{ounce_in} := 7.06155181 \cdot 10^{-3} \cdot \text{N} \cdot \text{m} \quad 1 \text{ Nm} = 141.612 \text{ ounce_in} \quad \text{milli} := 10^{-3} \quad \mu := 10^{-6}$$

$$\text{kg} \cdot \text{m}^2 = 141.612 \text{ ounce_in} \cdot \text{s}^2 \quad \text{krpm} := 104.72 \cdot \frac{\text{rad}}{\text{s}} \quad \text{rev} := 2 \cdot \pi \cdot \text{rad}$$

$$\text{kp} = \text{kgf} = \text{kilopond force} = \text{kg force} \quad 1 \frac{\text{rad}}{\text{s}} = 9.549 \times 10^{-3} \text{ krpm} \quad \text{rpm} := \frac{\text{krpm}}{1000}$$

$$\text{kp} := 35.274 \cdot \frac{\text{oz}}{\text{kg}} \quad \text{kgf} = 9.80665 \text{ N} \quad \text{N} = 0.101972 \text{ kgf}$$

$$\frac{2.2046 \cdot \text{lb}}{1 \cdot \text{kg}} = 1 \quad \frac{35.2736 \cdot \text{oz}}{1 \text{ kg}} = 1 \quad \text{rms} := 1 \quad \text{p} := 1 \quad \text{rms and peak abbreviations}$$

$$1 \cdot \frac{\text{N} \cdot \text{m}}{\text{A}} = 141.612 \frac{\text{ounce_in}}{\text{A}} \quad \text{Kt units}$$

$$1 \cdot \frac{\text{V}}{\frac{\text{rad}}{\text{s}}} = 104.72 \frac{\text{V}}{\text{krpm}} \quad \text{Ke units}$$

$$1 \cdot \frac{\text{ounce_in}}{\text{krpm}} = 6.743 \times 10^{-5} \frac{\text{N} \cdot \text{m}}{\frac{\text{rad}}{\text{s}}} \quad \text{D units (mech friction torque)}$$

Motor Example BEI DIP30-07-001A

Datasheet Parameters [See Motor Specifications on Page 6]

This poses a set of problems due to the fact that the KT and Ke values are ambiguous with respect to pk or rms. Also listed is $K_t = K_e$ ($0.226 \text{ V} / \text{rad} / \text{s} = 32 \text{ ounce_in} / \text{A}$). so we need to identify the proper units.

Sinusoidal Specifications

$$K_M := 13.5 \cdot \frac{\text{ounce_in}}{\sqrt{\text{W}}} \quad R_{\phi\phi} := 5.6 \cdot \Omega \quad P := 12 \quad \text{No. of magnetic poles}$$

$$L_{\phi\phi} := 8 \cdot \text{milli} \cdot \text{H} \quad J_M := 0.001923 \cdot \text{ounce_in} \cdot \text{s}^2$$

$$\tau_m := 1.49 \cdot \text{milli} \cdot \text{s} \quad \tau_3 := 1.43 \cdot \text{milli} \cdot \text{s}$$

$$\omega_0 := 106 \cdot \frac{\text{rad}}{\text{s}} \quad \text{Theoretical no load speed @ 24 V.}$$

$$\tau_p := 100 \cdot \text{ounce_in} \quad \text{Peak (stall) torque.}$$

Using (48), solving for K_{Trms}

$$K_{TrmsSS} := K_M \cdot \sqrt{\frac{3}{2}} \cdot \sqrt{R_{\phi\phi}}$$

$$K_{TrmsSS} = 0.276 \frac{N \cdot m}{A \cdot rms}$$

$$K_{TrmsSS} = 39.127 \frac{\text{ounce} \cdot \text{in}}{A \cdot rms}$$

Continuing with the parameter relationships:

and converting to SI Units

Verification Sine / Sine row

Attachment A, Chart 1, Col 2

$$K_{TpSS} := \frac{K_{TrmsSS}}{\sqrt{2}}$$

$$K_{TpSS} = 0.195 N \cdot \frac{m}{A \cdot p}$$

$$K_{e\phi\phi p} := \sqrt{\frac{2}{3}} \cdot K_{TrmsSS}$$

Attachment A, Chart 1, INV(Col 1*Col 2)

$$K_{e\phi\phi p} = 0.226 \frac{V}{\frac{\text{rad}}{s}}$$

So the $K_{e\phi\phi p}$ appears to be the correct value for back emf.

$$K_{TpSS} := \frac{\sqrt{3}}{2} \cdot K_{e\phi\phi p}$$

Attachment A, Chart 1, Col 1

$$K_{TpSS} = 0.195 \frac{N \cdot m}{A \cdot p}$$

$$K_{e\phi\phi rmsSS} := \frac{K_{TrmsSS}}{\sqrt{3}}$$

Attachment A, Chart 1, Col 3

$$K_{e\phi\phi rmsSS} = 0.16 \frac{V \cdot rms}{\frac{\text{rad}}{s}}$$

$$K_{e\phi\phi p} := \sqrt{2} \cdot K_{e\phi\phi rmsSS}$$

Attachment A, Chart 1, Col 4

$$K_{e\phi\phi p} = 0.226 \frac{V \cdot p}{\frac{\text{rad}}{s}}$$

$$K_{MSS} := K_M$$

$$J_M = 1.358 \times 10^{-5} \text{ kg m}^2$$

$$K_{MSS} = 0.0953 \frac{N \cdot m}{\sqrt{W}}$$

$$K_{e\phi\phi pSS} := K_{e\phi\phi p}$$

Though $K_{e\phi\phi p}$ is mostly used.

Figures of Merit - Calculated

Stall Torque

$$V_{\phi\phi\text{ptest}} := 17.4 \text{ V} \cdot \text{p}$$

(100)
"Page 13"

$$\tau_{\text{stallSS}} := \frac{2}{\sqrt{3}} \cdot \frac{V_{\phi\phi\text{ptest}}}{R_{\phi\phi}} \cdot K_{\text{TpSS}} \quad \text{From (40)} \quad I_{\phi\phi\text{pstall}} := \frac{2}{\sqrt{3}} \cdot \frac{V_{\phi\phi\text{ptest}}}{R_{\phi\phi}} \quad I_{\phi\phi\text{pstall}} = 3.588 \text{ A}$$

$$I_{\phi\phi\text{pstall_spec}} = \frac{V_{\phi\phi\text{ptest}}}{R_{\phi\phi}} = 3.107 \text{ A} \cdot \text{p} \quad \text{Incorrect equation (41) used in Spec.! Use for Squarewave drive only.} \quad I_{\phi\phi\text{rmsstall}} := \frac{I_{\phi\phi\text{pstall}}}{\sqrt{2}} \quad I_{\phi\phi\text{rmsstall}} = 2.537 \text{ A} \cdot \text{rms}$$

$$\tau_{\text{stallSS}} = 0.701 \text{ N} \cdot \text{m} \quad \tau_{\text{stallSS}} = 99.263 \text{ ounce_in} \quad \text{Check Spec states 100 ounce_in for Tp}$$

No load speed @ 24 V

$$V_{\phi\phi\text{ptest}} := 24 \cdot \text{V} \cdot \text{p}$$

Using (42)

$$\omega_{\text{NL}} := \frac{V_{\phi\phi\text{ptest}}}{K_{\phi\phi\phi\text{p}}} \quad \omega_{\text{NL}} = 106.386 \frac{\text{rad}}{\text{s}} \quad \text{Checks with spec. } (\omega_0)$$

Speed-Torque Curve

$$(42) \quad \omega_{\text{NL}} = \frac{V_{\phi\phi\text{p}}}{K_{\phi\phi\phi\text{p}}} = \frac{V_{\phi\phi\text{rms}}}{K_{\phi\phi\phi\text{rms}}}$$

No load test values:

$$V_{\phi\phi\text{p}} := 17.4 \cdot \text{V} \cdot \text{p} \quad \omega_{\text{NL1}} := \frac{V_{\phi\phi\text{p}}}{K_{\phi\phi\phi\text{p}}} \quad (42) \quad \omega_{\text{NL1}} = 77.13 \frac{\text{rad}}{\text{s}}$$

$$V_{\phi\phi\text{rms1}} := V_{\phi\phi\text{p}} \cdot \frac{\text{rms}}{\sqrt{2}} \quad \omega_{\text{NL1}} := \frac{V_{\phi\phi\text{rms1}}}{K_{\phi\phi\phi\text{rmsSS}}} \quad \omega_{\text{NL1}} = 77.13 \frac{\text{rad}}{\text{s}} \quad \text{Checks}$$

Torque test values

$$\begin{aligned} \tau_1 &:= 0 \cdot \text{ounce_in} & \tau_1 &= 0 \text{ N} \cdot \text{m} \\ \tau_2 &:= 35.00 \cdot \text{ounce_in} & \tau_2 &= 0.247 \text{ N} \cdot \text{m} \\ \tau_3 &:= 90 \cdot \text{ounce_in} & \tau_3 &= 0.636 \text{ N} \cdot \text{m} \end{aligned}$$

Speed – Torque slope

Using (44)

$$R_{\text{mSS}} := \frac{\sqrt{3}}{2} \cdot \frac{R_{\phi\phi}}{K_{\phi\phi\phi\text{rmsSS}} \cdot K_{\text{TrmsSS}}} \quad R_{\text{mSS}} = 110.035 \frac{\frac{\text{rad}}{\text{s}}}{\text{N} \cdot \text{m}}$$

Speed-Torque test points

Using (46)

$$\omega_1 := \omega_{NL1} - R_{mSS} \cdot \tau_1 \quad \omega_1 = 77.13 \frac{\text{rad}}{\text{s}}$$

$$\omega_2 := \omega_{NL1} - R_{mSS} \cdot \tau_2 \quad \omega_2 = 49.934 \frac{\text{rad}}{\text{s}}$$

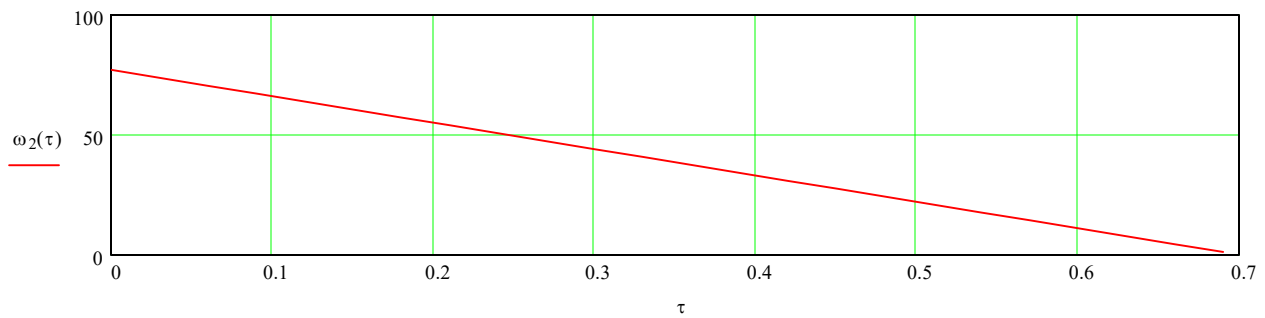
$$\omega_3 := \omega_{NL1} - R_{mSS} \cdot \tau_3 \quad \omega_3 = 7.198 \frac{\text{rad}}{\text{s}}$$

$$\tau := 0 \cdot \text{N} \cdot \text{m}, (0.03 \cdot \text{N} \cdot \text{m}) .. 0.7 \cdot \text{N} \cdot \text{m}$$

define range variable

Plot ω_2 vs τ

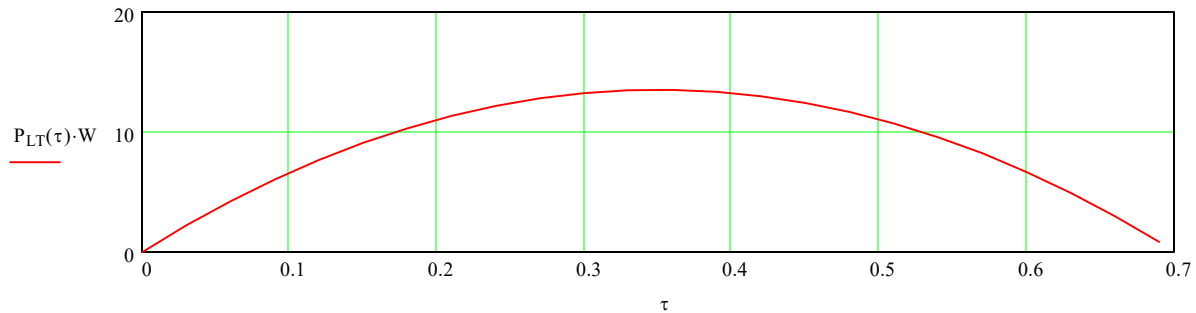
Speed Torque Curve $\omega_2(\tau) := \omega_{NL1} - R_{mSS} \cdot \tau$ Equation (46)



$$\omega_2(0 \cdot \text{N} \cdot \text{m}) = 77.13 \frac{\text{rad}}{\text{s}} \quad \omega_2\left(\frac{\tau_{\text{stallSS}}}{2}\right) = 38.565 \frac{\text{rad}}{\text{s}} \quad \omega_2(\tau_{\text{stallSS}}) = -0 \frac{\text{rad}}{\text{s}}$$

Load Power as a function of torque:

$$P_{LT}(\tau) := -\left(\frac{\omega_{NL1}}{\tau_{\text{stallSS}}}\right) \cdot \tau^2 + \omega_{NL1} \cdot \tau$$

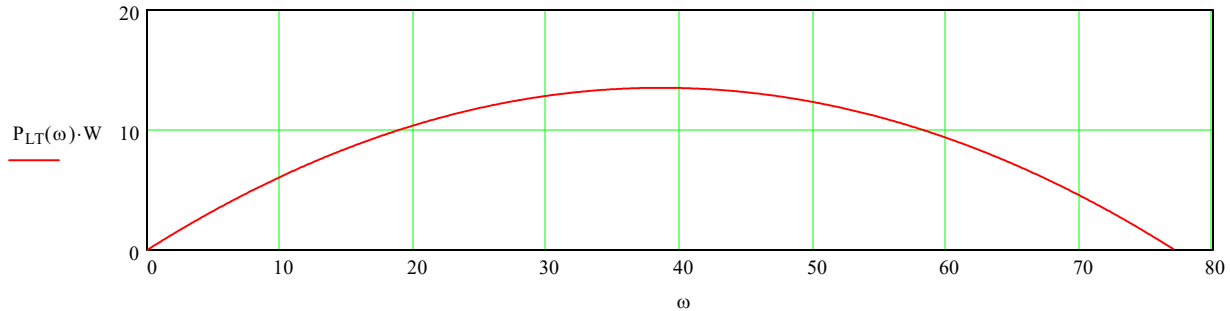


$$P_{LT}\left(\frac{\tau_{\text{stallSS}}}{2}\right) = 13.516 \text{ W}$$

$$\omega := 0 \cdot \frac{\text{rad}}{\text{s}}, \left(0.1 \cdot \frac{\text{rad}}{\text{s}} \right) .. 77 \cdot \frac{\text{rad}}{\text{s}} \quad \text{Range variable defined}$$

Load Power as a function of speed

$$P_{LT}(\omega) := - \left(\frac{\tau_{\text{stallSS}}}{\omega_{NL1}} \right) \cdot \omega^2 + \tau_{\text{stallSS}} \cdot \omega$$



$$P_{LT} \left(\frac{\omega_{NL1}}{2} \right) = 13.516 \text{ W}$$

Using (27)

$$P_R := \frac{3}{2} \cdot \left(\frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TrmsSS}} \right)^2 \cdot R_{\phi\phi} \quad \text{with } I_{\phi\phi} \text{ argument}$$

$$I_{\phi\phi\text{rmsStall}} := \left(\frac{\tau_{\text{stallSS}}}{K_{TrmsSS}} \right)$$

$$P_R = 13.516 \text{ W}$$

$$I_{\phi\phi\text{rmsStall}} = 2.537 \text{ A}$$

Using (14)+(15)

$$P_T := \frac{\omega_{NL1}}{2} \cdot \frac{\tau_{\text{stallSS}}}{2} + \frac{3}{2} \cdot R_{\phi\phi} \cdot \left(\frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TrmsSS}} \right)^2$$

$$P_{LT} \left(\frac{\omega_{NL1}}{2} \right) + P_R = 27.032 \text{ W}$$

$$P_T = 27.032 \text{ W}$$

Checks

Motor Constant

Using (48)

$$K_{MSS} := \sqrt{\frac{2}{3}} \cdot \frac{K_{TrmsSS}}{\sqrt{R_{\phi\phi}}} \quad K_{MSS} = 0.095 \frac{\text{N}\cdot\text{m}}{\sqrt{\text{W}}}$$

Also

$$K_{MSS} := \frac{2}{\sqrt{3}} \cdot \frac{K_{TpSS}}{\sqrt{R_{\phi\phi}}} \quad K_{MSS} = \frac{\text{N}\cdot\text{m}}{\sqrt{\text{W}}}$$

Time constants

Using (52)

$$\tau_m := \frac{\sqrt{3}}{2} \cdot \frac{R_{\phi\phi} \cdot J_M}{K_{c\phi\phi p} \cdot K_{TpSS}}$$

Using (54)

$$\tau_e := \frac{L_{\phi\phi}}{R_{\phi\phi}}$$

Note No difference in either time constants published or caculated!

$$\tau_m = 1.494 \text{ milli}\cdot\text{s} \quad \text{Checks}$$

$$\tau_e = 1.429 \text{ milli}\cdot\text{s} \quad \text{Checks}$$

$$K_{TpSS} \cdot K_{c\phi\phi p} = 0.044 \Omega \cdot \frac{\text{N}\cdot\text{m}}{\frac{\text{rad}}{\text{s}}}$$

From Attachment A, Chaart 1 Col 5.

Both Check

From Attachment A, Chaart 2 Col 10.

$$\frac{K_{TrmsSS}^2}{\sqrt{3}} = 0.044 \Omega \cdot \frac{\text{N}\cdot\text{m}}{\frac{\text{rad}}{\text{s}}}$$

$$K_{TrmsSS} \cdot K_{c\phi\phi\text{rmsSS}} = 0.044 \Omega \cdot \frac{\text{N}\cdot\text{m}}{\frac{\text{rad}}{\text{s}}}$$

Resistor power calculation using the Motor Constant, KM.

$$P_R = \left(\frac{\tau}{K_M} \right)^2 \quad \text{Derived from definition (102)} \quad \text{"Page 15"}$$

$$P_R := \left(\frac{\frac{\tau_{\text{stallSS}}}{2}}{K_M} \right)^2$$

$$P_R = 13.516 \text{ W} \quad \text{Checks}$$

Load power equation verifications

$$P_{LT} = \frac{\omega_{NL1}}{2} \cdot \frac{\tau_{\text{stallSS}}}{2} = 13.516 \text{ W} \quad (\tau \cdot \omega)$$

From (25a)

$$P_{LTSS} := \frac{\sqrt{3}}{2} \cdot K_{\phi\phi p} \cdot \left(\frac{\omega_{NL1}}{2} \right) \cdot \left(\frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TpSS}} \right)$$

with $I_{\phi\phi p}$ calculation

$$I_{\phi\phi pSS} := \frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TpSS}}$$

$$P_{LTSS} = 13.516 \text{ W}$$

Check

$$I_{\phi\phi pSS} = 1.794 \text{ A}\cdot\text{p}$$

From (25b)

$$P_{LTSS} := \sqrt{3} \cdot K_{\phi\phi rmsSS} \cdot \left(\frac{\omega_{NL1}}{2} \right) \cdot \left(\frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TrmsSS}} \right)$$

with $I_{\phi\phi rms}$ calculation

$$I_{\phi\phi rmsSS} := \frac{\frac{\tau_{\text{stallSS}}}{2}}{K_{TrmsSS}}$$

$$P_{LTSS} = 13.516 \text{ W}$$

Check

$$I_{\phi\phi rmsSS} = 1.268 \text{ A}\cdot\text{rms}$$

$$\frac{I_{\phi\phi rmsSS}}{I_{\phi\phi pSS}} = \frac{1}{\sqrt{2}} \quad \text{Verifies (A11)}$$

DWG NO.

DIP30-07-001A

WINDING CONSTANTS *	UNITS	TOL	SYM	WDG A
DC RESISTANCE	OHMS	±12.5%	R	5.6
VOLTAGE @ T_p	VOLTS	NOMINAL	V_p	17.4
CURRENT @ T_p	AMPERES	NOMINAL	I_p	3.1
TORQUE SENSITIVITY	OZ-IN/AMP	±10%	K_T	32.0
BACK EMF CONSTANT	VOLTS/(RAD/SEC)	±10%	K_B	0.226
INDUCTANCE @ 1 KHz	MILLIHENRY	±30%	L	8.0

MOTOR PARAMETERS *	UNITS	SYM	NOM. VALUE
PEAK TORQUE	OZ-IN	T_p	100
MOTOR CONSTANT	OZ-IN/ $\sqrt{\text{WATT}}$	K_M	13.5
ELECTRICAL TIME CONSTANT	MILLISECOND	τ_E	1.43
MECHANICAL TIME CONSTANT	MILLISECOND	τ_M	1.49
POWER $I^2 R$ @ T_p	WATTS	P_p	53.8
DAMPING FACTOR (ZERO IMPEDANCE)	OZ-IN/(RAD/SEC)	F_0	1.29
ROTOR INERTIA	OZ-IN SEC ²	J_M	1.923×10^{-3}
THEO. NO-LOAD SPEED @ 24 VDC	RAD/SEC	ω_0	106
THEO ACC @ T_p	RAD/SEC ²	α_T	5.2×10^4
THERMAL RESISTANCE	°C/WATT	θ_{TH}	5.2
MAX ALLOWABLE WINDING TEMP	°C	TEMP	155
NUMBER OF PHASES/WINDING TYPE			3/Y
NUMBER OF POLES			12
WEIGHT	OZ	W_T	7.5

* 25°C AMBIENT

DIRECTION OF ROTATION

CW VIEWED FROM LEAD SIDE

